



Development and Performance Evaluation of a Livestock Feed Chopper/Pulverizer

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ABSTRACT: Crop residues are bulky and low in nutrients. Their size reduction is imperative in order to compound them with other feed ingredients to achieve balanced feedstock for livestock. A chopping and pulverizing machine was designed and constructed to chop and pulverized feed materials. It comprises of chopping hopper, pulverizing hopper, metering device, chopping/ pulverizing chamber, screen and the processed feed outlet. The performance of the machine was evaluated on the basis of throughput capacity, pulverizing efficiency, scatter losses as dependent variables while moisture content was the independent variable. There were two level of speeds (1500 rpm and 2100 rpm) and five levels of moisture content (13 %, 16 %, 19 %, 21 %, 24 %, and 27 % dry basis). The throughput capacity, pulverizing efficiency had highest values of 222 kg/h and of 92.5 % respectively at moisture content level of 13 % (db) and 2100 rpm speed. However, the least scatter losses obtained was 2.5 % at moisture content level of 27 % (db) and 1500 rpm speed. As the moisture content increased, throughput capacity, pulverizing efficiency and scatter losses decreases at both speeds. The developed machine could enhance the use of crop residues in feeding livestock which could ultimately reduce the cost of livestock feed production.

DOI:<https://dx.doi.org/10.4314/jasem.v25i9.14>

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Dates: Received: 09 May 2021; Revised: 12 August 2021; Accepted: 12 September 2021

Keywords: Chopping, Pulverization, Livestock, Throughput capacity

Livestock farmers in the tropics often have to contend with the problems of getting balanced diet for their animals during the dry season when crop residues are low in quantity and quality (Kellems and Church, 1998). Satisfying the nutritional needs of animals during the dry season is a serious issue as inadequate feeding of livestock limits their productivity in terms of growth and milk yield which ultimately diminishes profit of the farmer. Underfed animals also are not agile and prone to disease and reduction in fertility. In urban areas, natural grassland scarce and farmer depend on grown pastures, crop residues, agro industrial by-product to feed their animals. Silva *et al.* (2008) reported that due to the seasonal availability of crop residues and fodders in urban areas, the scarcity of agro industrial by-product and the high prices of formulated feeds would make it difficult to fulfil the dietary needs of the expanding animal population. To tackle the limiting supply of livestock feed, farmers rely on crop residues for their feed supply. Stovers are dry stalks from mature maize or guinea corn, without grains. Cereal stovers are popular for feeding livestock. In the semi-arid areas guinea corn and maize stovers are vital part of ruminant animal feeds during the dry season which help to sustain feed supply and survival (Sibanda, 1986). However, cereal stovers are bulky, fibrous and relatively poor in nutritive value. Likewise, low ease of digestion and bulkiness of

stovers, the poor crude protein quality and the lack of other vital nutrients limit its use in the diets of productive animals (Dzowela, 1987). Dzowela (1987) asserted that crop residues could be augmented with brans and millings, oilseed cakes, legumes and fodder from multipurpose trees. Experimental results with sheeps reported by Osafo *et al.* (1993) where chopped sorghum stover combined with mineral lick was sufficient to sustain the weight of rams between the age 15 and 20 months. It was claimed that chopped sorghum stover with generous amount of minerals and no other supplement was a sustainable dietary strategy for maintaining weight in sheep are close to maturity. Crop residue could be managed by employing processing technologies for the production of balanced straw-based feed for animals (Walli *et al.*, 2012). Physical treatment includes chopping, shredding, grinding and pelletization. Abayineh (2019) adapted two hammer mills for crushing crop residues in order to make livestock feeds. The crushing efficiency obtained ranged from 94 to 99.33 % while the throughput capacity ranged from 78.68 to 152.54 kg/h. There were indications that size reduction and pelletization of fibrous materials increased the exposed surface area which accelerate the digestibility in the intestines of livestock. Also, size reduction and pelletization results in more than 30% increase in feed intake (Hoque *et al.*, 2007). Lyu *et al.* (2020) asserted

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that size reduction of feed ingredients further enhances the ability of livestock to digest the nutrients in the feeds. The bulkiness and low nutrients content of the crop residues necessitate size reduction in order to be compounded with other feed ingredients to achieve balanced feedstock for livestock. Therefore, the aim of this study was to design, fabricate and investigate the performance of a combined feed chopping and pulverizing machine for processing crop residue for ease of pellet formation from them.

MATERIALS AND METHODS

Description of the feed chopper/pulverizer: The livestock feed chopper/pulverizer was designed and fabricated (figure 1). It comprises of chopping hopper, pulverizing hopper, metering device, chopping/pulverizing chamber, screen and the processed feed outlet. The two hoppers were trapezoidal in shape. They were raised 30° above the horizontal plane to aid the free flow of feed materials (stovers and grasses) in the machine. The chopping/pulverizing chamber comprised of a chopping section which has sharp edged rectangular steel blades for cutting fed materials into desired length (sizes) and a pulverizer which has swinging steel hammers mounted on a rotating shaft. The hammers continuously beat and breakdown the materials in the pulverizer until they are small enough go through the apertures of the screen. The screen is a detachable curved metal screens for passage of pulverised materials. The processed feed outlet was the point of collection of the pulverized materials. The chopping section had no screen below it.

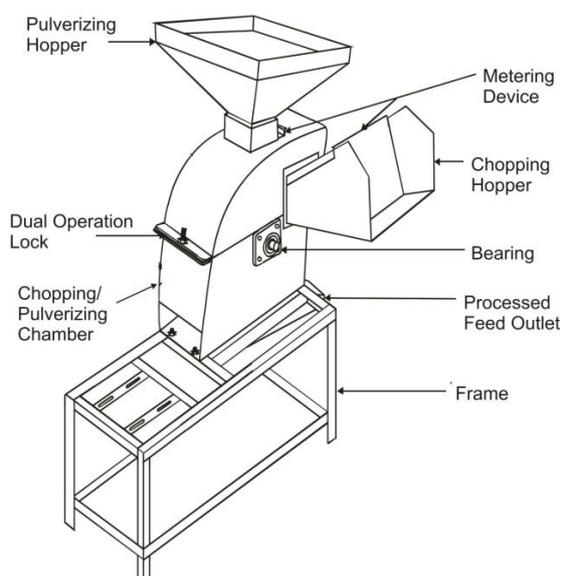


Fig 1. Isometric View of the Livestock Feed Chopper/Pulverizer

Design of the components of the machine: In designing the machine, availability and strength of the selected

fabrication materials to withstand the various loads from the operation of the machine were duly considered. The angle of repose of the feed materials were also considered in the configuration of the hoppers. Likewise, arrangement of the chopping knives, pulverizing hammers and aperture of screen to give desired material sizes were considered. In order to design the hammers inside the pulverizing chamber, equation (1) given by Khurmi and Gupta (2005) was used

$$\frac{M_b}{I} = \frac{\sigma_b}{y} \quad (1)$$

Where M_b is Maximum bending moment of the rectangular hammer (Nm); I is moment of inertia (m^4); σ_b is Allowable stress on the steel hammer 250 MPa (Khurmi and Gupta, 2005); y is distance of the outermost layer of the bar from the neutral axis (m). The dimensions of the hammer were designed to length of 0.06m, width of 0.008m and thickness of 0.004 m.

The cutting force of the rotary knives determined by applying equation (2) as given by Ebunilo *et al.* (2010)

$$F_c = \frac{M_T \omega^2 D_K}{2} \quad (2)$$

Where F_c is cutting force (N); M_T is total mass of the two knives (kg); D_K is length of the knife (m); ω is angular velocity of knife (rad/s). The cutting force of the knife was calculated to be 129 N

The nominal belt pitch length was computed with the formula given by Khurmi and Gupta (2005):

$$L = 2C + \frac{\pi}{2}(D_1 + D_2) + \left[\frac{(D_2 - D_1)^2}{4C} \right] \quad (3)$$

Where D_1 is of motor pulley diameter; D_2 is of chopper/pulverizer shaft pulley diameter; C is center to center distance between electric motor and chopper/pulverizer pulley. From the design, standard A50 was chosen.

According to Khurmi and Gupta (2005), for a solid shaft which has little or no axial load, the shaft diameter was calculated by:

$$d^3 = \frac{16}{\pi S_s} \times \sqrt{(K_b M_b)^2 + (K_t M_t)^2} \quad (4)$$

Where d is the shaft Diameter (m); S_s is Allowable stress ($40 \times 10^6 Nm^{-2}$) (Hall *et al.*, 1980); K_b is Combine shock and fatigue factor applied to bending

moment (1.0) (Hall *et al.*, 1980); M_b is bending moment (Nm); K_t is Combine shock and fatigue factor applied to torsional moment (1.0); M_t is Torsional moment (Nm). The computed diameter of the shaft was 38 mm. The detailed dimensions of the machine are as shown in figure 2.

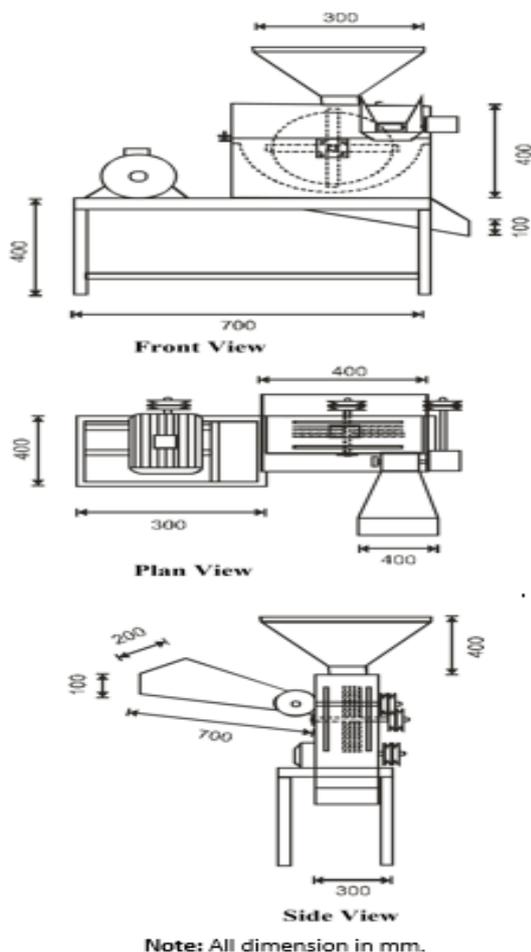


Fig 2. The Orthographic view of the machine

Principle of operation of the machine: The fabricated machine (figure 3) was operated by collecting crop residues like stover and feeding them into the chopping section of the machine. As the materials entered through the hopper, the high-speed rotating knives cut the materials into predetermined lengths of chop and were immediately collected below the machine. The chopped materials were further dried to attain low moisture contents which would enhance the pulverisation process. The dried chopped materials were fed into the pulverizing chamber through the pulverizing hopper. The high-speed rotating hammers pulverized the by impacting blows on materials inside the chamber. The materials were beaten until it could pass through the apertures of the screen. At this point other materials like wheat bran, cassava peels and

groundnut cake needed to be added to the livestock feed are introduced for pulverization. Materials were continuously beaten by the rotating hammers until the materials could pass through the screen perforations. The processed feed outlet is an inclined trapezoidal shaped trough made from mild steel sheets placed directly below the perforated screen of the chopping/pulverizer chamber to collect the pulverised feeds that passed through the screen section.



Fig 3. The Pictorial view of the fabricated machine

Procedure for the performance evaluation of the machine: Two feed stocks (elephant grass and corn stover) were obtained. They were chopped using the chopping section of the machine to an average uniform length of 20 mm and were sun-dried until moisture content value was 13% dry basis (db). 500 g of elephant grass and 500 g of corn stover were thoroughly mixed to form samples of 1000 g. The machine was tested at two level of speeds (1500 rpm and 2100 rpm) and five levels of moisture content (13 %, 16 %, 19 %, 21 %, 24 %, and 27 % dry basis). Different pulley sizes were used to achieve the two speeds. In order to achieve the levels of moisture content, samples of 1000g were conditioned by addition of calculated amount of water to increase the moisture content to the various levels. Equation (5) given by Adejumo and Abayomi (2012) was used to determine the amount of water to be added to achieve the various moisture levels. After the addition of the right quantity of water each sample was sealed in nylon bags and left for three hours for the purpose of even moisture distribution and stability (Sharma *et al.*, 2013).

$$Q = W_i \frac{(M_f - M_i)}{(100 - M_f)} \quad (5)$$

Where Q is the quantity of water (g); W_i is initial mass of sample (g); M_f is the final moisture content (% db); M_i is initial moisture content of sample (% db).

The initial moisture content of feedstock was found to be 13% (db). The machine was tested by the application of completely randomized design. The conditioned samples were used for the performance evaluation. Test at each moisture level was replicated three times. The screen size used was 3 mm. Measurements were carried out for each run of the experiment. The performance indices used to evaluate the machine were throughput capacity, pulverizing efficiency and scatter losses

Determination of throughput capacity of machine:

This is the ratio of the weight of feed materials reduced (pulverized) to the total time taken to complete the size reduction operation expressed in kg/h and given as:

$$C = \frac{W}{t} \quad (6)$$

Where C is Throughput capacity of the machine (kg/h); W is Weight of the reduced feed material (kg); t is Total time taken to carry out the size reduction operation (h)

Determination of pulverizing efficiency of the machine: This is the ratio of the quantity of feed materials reduced to the total quantity of feed materials fed into the hopper expressed as a percentage.

$$P_e = \frac{Q_g}{Q_t} \times 100 \quad (7)$$

Where P_e is the Pulverizing efficiency (%); Q_g is quantity of feed materials reduced (kg); Q_t is total quantity of feed materials fed into the hopper (kg)

Determination of scatter losses of the machine: During the size reduction operation, some reduced materials could be lost to scattering. Amount of such reduced materials were determined using the formula below and expressed as a percentage;

$$S_L = \frac{Q_L}{Q_t} \times 100 \quad (8)$$

Where S_L is the scatter losses (%); Q_L is Quantity of feed materials scattered around the machine after size reduction (kg); Q_t is Total quantity of feed material in sample (kg)

RESULTS AND DISCUSSION

Performance of the machine: The relationship between the dependent variables which were throughput capacity, pulverizing efficiency, scatter

losses and independent variable which was moisture content was used to investigate the performance of the machine. A fixed screen size of 3mm and machine speeds of 1500 rpm and 2100 rpm were used throughout the investigation.

Effect of Moisture Contents on Throughput capacity:

Figure 4 shows the effect of moisture content on throughput capacity of the machine at operating speeds of 1500 rpm and 2100 rpm. The moisture content was significant at 5% level. The figure also shows that the maximum throughput capacity at moisture content value of 13% (db) were 204 kg/h and 222 kg/h at 1500 rpm and 2100 rpm operating speed respectively. The minimum throughput capacity at moisture content value of 27% (db) at were 166 kg/h and 176 kg/h at 1500 rpm and 2100 rpm operating speed respectively. It was observed that the throughput capacity decreased as the moisture content increased at both speeds. This could be due to the fact that at low moisture content it was easier to pulverize the feed material because of being crispy in nature thereby leading to high throughput capacity. This result follows the same trend reported by Abayineh (2019) where it was claimed that as the moisture content of maize stalk increased the throughput capacity of the machine decreased.

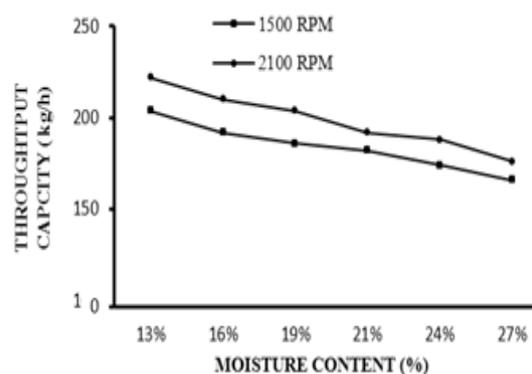


Fig 4. Effect of moisture content on the throughput capacity at two speeds

Effect of Moisture Contents on Pulverizing efficiency:

Figure 5 shows the effect of moisture content on the pulverizing efficiency of the machine at operating speeds of 1500 rpm and 2100 rpm. The moisture content was significant at 5% level. The figure also depicts that the maximum pulverizing efficiency at moisture content level of 13% (db) were 85.5% and 92.5% at 1500 rpm and 2100 rpm operating speed respectively. The minimum pulverizing efficiency at moisture content level of 27% (db) were 67.5% and 72.5% at 1500 rpm and 2100 rpm operating speed respectively. It was observed that the pulverizing efficiency decreased as the moisture content increased

at both speeds. This could be due to the fact that at high moisture, the feedstock could be elastic thereby making it difficult to be reduced by the hammers. This result is in similar trend with that of Idris et al. (2018) where it was claimed that as the moisture content of paddy rice increased the pulverizing efficiency of the crop thresher reduced.

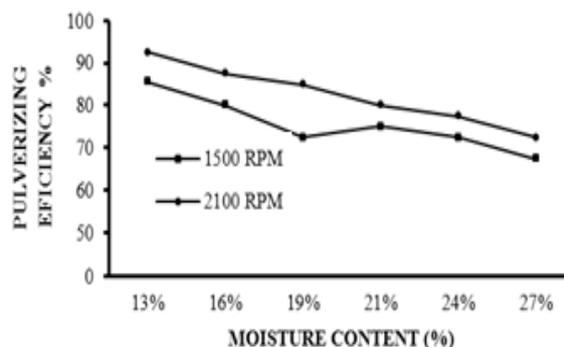


Fig 5. Effect of moisture content on the pulverizing efficiency at two speeds

Effect of Moisture Contents on Scatter Losses: Figure 6 shows the effect of moisture content on the scatter losses of the machine for the operating speeds of 1500 rpm and 2100 rpm. Moisture content was observed to be significant at 5% level. Also, the figure shows that the maximum percentage scatter loss at moisture content value of 13% (db) were 10% and 7.5% at 1500 rpm and 2100 rpm operating speed respectively.

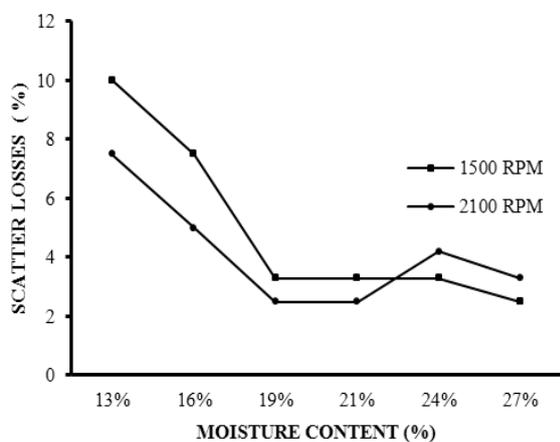


Fig 6. Effect of moisture content on the scatter losses at two speeds

The minimum percentage scatter loss at moisture content value of 27% (db) were 2.5% and 3.3% at 1500 rpm and 2100 rpm operating speed respectively. The graph depicts that the percentage scatter losses decreased as the moisture content increased at both speeds. This may be due to the fact that at low moisture content of the feedstock, the air from the hammers

blows off the pulverized feedstock resulting to increase in scatter losses. Baryeh (2002) and Ojediran et al. (2010) reported that the dimensions of millet seed grain increased with the increase in their moisture content thereby becoming more difficult to be blown off. Probst et al. (2001) reported that the loss of smaller particles of ground corncobs decreased as moisture content increases due to particle cohesion.

Conclusion: A livestock feed chopper/pulverizer was designed and fabricated for livestock feeds. The performance of the machine was determined in terms of throughput capacity, pulverizing efficiency and scatter losses as dependent variables while moisture content was the independent variable. The highest throughput capacity of 222 kg/h and pulverizing efficiency of 92.5% were obtained while the least scatter losses obtained was 2.5%. The developed machine could enhance the use of crop residues which hitherto were wasted.

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